

②

EUROPEAN PATENT APPLICATION

②' Application number: 88305067.6

⑤' Int. Cl.4: B60T 10/02

② Date of filing: 03.06.88

③ Priority: 23.06.87 US 65350

④ Date of publication of application:
28.12.88 Bulletin 88/52

④ Designated Contracting States:
BE CH DE FR GB IT LI NL SE

⑦ Applicant: GENERAL MOTORS CORPORATION
General Motors Building 3044 West Grand
Boulevard
Detroit Michigan 48202(US)

⑦ Inventor: Fuehrer, Reece R.
51 North 200 East
Danville Indiana 46122(US)
Inventor: Won, Harvey Donald
786 Winter Court
Carmel Indiana 46032(US)

⑦ Representative: Denton, Michael John et al
Patent Section - Luton Office (F6) Vauxhall
Motors Limited P.O. Box 3 Kimpton Road
Luton Bedfordshire LU2 0SY(GB)

⑤ Control circuit for vehicle retardation.

⑤ A control circuit for vehicle retardation has a hydraulic retarder (26) and a slipping friction brake (14) operating in concert to retard the forward motion of the vehicle. The hydraulic retarder is an output driven device which is operable to absorb increasing energy as the vehicle speed increases. The friction brake is normally a ratio control reaction brake for the lowest forward drive ratio of the vehicle transmission. The friction brake has an additional apply piston (84,86) which is operable to slippingly engage the friction brake whenever the hydraulic retarder is operable and the vehicle transmission is not in the lowest forward ratio. The control circuit has a control valve (52) which is operable to control the engagement pressure of the additional piston in a manner such that the friction braking effort is higher at low vehicle speeds. This compensates for the lower effectiveness of the output driven hydraulic retarder. The control circuit is also effective to control the operation of the hydraulic retarder, the main piston of the reaction brake, cooling flow to the friction brake and cooling flow from the hydraulic retarder.

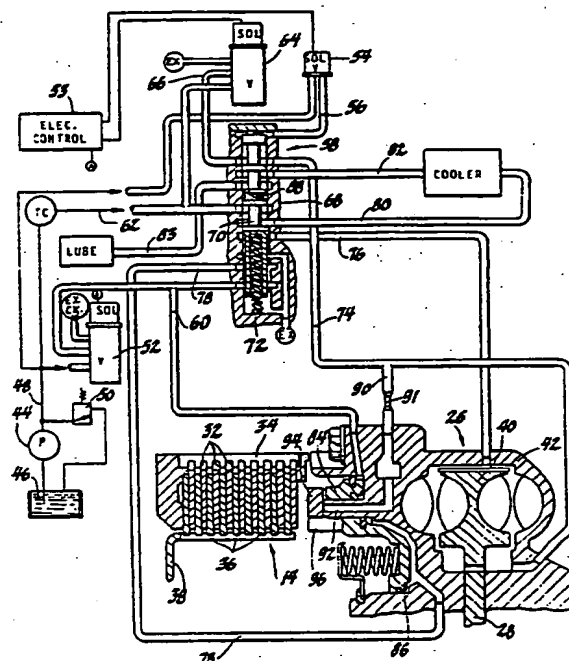


Fig. 2

CONTROL CIRCUIT FOR VEHICLE RETARDATION

This invention relates to a control circuit for vehicle retardation for a vehicle having hydraulic transmission, and more particularly a control circuit having both hydraulic and friction brake retardation.

Vehicle retardation arrangements which employ the vehicle service brakes and the hydraulic retarder in concert have been utilized in the past. These arrangements generally use the service brake only when the rapid retardation is required to bring the vehicle to a stop. During normal retardation, only the hydraulic retarder is in use since continued slipping of the vehicle service brakes causes significant heat generation and brake wear. It has also been proposed to control the transmission drive ratio during retardation. In these latter arrangements (as for example shown in US Patent No 4630507), the transmission is shifted to a lower drive ratio so that the torque converter is driven at a high speed to maximize the engine braking; and an input driven retarder is also driven at a very high speed to improve the braking efficiency of the retarder.

It is an object of the present invention to provide an improved control circuit for vehicle retardation wherein a reaction brake for the lowest of the forward drive ratios is slippingly applied in concert with an output driven hydraulic retarder.

To this end, a control circuit in accordance with the present invention is characterised by the features specified in the characterising portion of Claim 1.

The present invention provides a control circuit which permits the use of a small diameter output driven hydraulic retarder while maintaining satisfactory retardation at low vehicle speeds. This is accomplished without adding to the outer diameter of the hydraulic retarder. The transmission reaction brake, of the lowest drive ratio, is used as a slipping friction brake. This permits improved retardation at low vehicle speeds when the vehicle transmission is in the second or higher gear ratio. As will be recognized by those familiar with the art, partial application of the low ratio brake tends to establish a lockup condition in the transmission. Since the friction brake is only slippingly applied, a transmission lockup condition does not occur while vehicle braking does. Normally vehicle retardation is not of great importance when the vehicle is operating in the lowest drive ratio or in reverse. That is why the low ratio rate has been selected as the vehicle retardation brake with this control circuit.

When the friction braking is used for low speed retardation, the output driven hydraulic retarder can be maintained in a smaller package. This is possi-

ble because, at the lower vehicle speeds, the friction retardation will be generally sufficient for slowing the vehicle when hydraulic retardation or hydraulic braking is not as important. With this arrangement, the apply pressure for the reaction brake is controlled so that complete brake apply cannot occur, and the apply pressure is reduced at higher vehicle speeds. Also, a portion of the retarder supply or feed oil is used to provide cooling and lubrication for the slipping brake. This oil is supplied only during retardation and not during the normal function of the reaction brake. At higher vehicle speeds, the hydraulic retarder has increased capacity, as is well known, and therefore the friction braking capacity can be reduced.

This invention also provides an improved control circuit in a vehicle having a multispeed power transmission with at least the lowest forward drive ratio being established by reaction brake, wherein the reaction brake has two apply pistons with only one of said pistons being pressurized during vehicle retardation so that at low vehicle speeds a slipping friction retarder acts in concert with a hydraulic retarder and further wherein the apply pressure of the one piston is reduced as vehicle speed is increased.

The present invention will now be described, by way of example, with reference to the following description and the accompanying drawings, in which:-

Figure 1 is a schematic representation of a planetary transmission incorporating a control circuit for retardation in accordance with the present invention;

Figure 2 is a diagrammatic representation of a portion of a hydraulic retarder and friction brake of Figure 1, and a hydraulic control circuit for controlling these devices; and

Figure 3 is a plot of output speed versus braking torque showing the various braking components and their summation.

The planetary gearing arrangement shown in Figure 1 is substantially identical with that described in U.S. Patent No. 4,070,927. This planetary transmission provides six forward speeds and one reverse speed through the use of two friction clutches 10 and 12 and three friction brakes 14, 16 and 18 which are operable to control three planetary gear sets 20, 22 and 24. The friction clutch 10 is engaged in the first or lowest speed ratio and also in the second, third and fourth speed ratios. The friction clutch 12 is engaged in the fourth, fifth and sixth speed ratios. Friction brake 14 is engaged in the first and reverse speed ratios. Friction brake 16 is engaged in the second and sixth speed

ratios and friction brake 18 is engaged in the third, fifth and reverse speed ratios. A hydraulic retarder generally designated 26 is drivingly connected through a hub 28 to an output shaft 30 of the planetary transmission. Friction brake 14 has a plurality of friction discs 32 which are connected to a stationary component 34 in the planetary transmission and a plurality of plates 36 which are drivingly connected to a power transmission member 38. For those wishing a more complete description of the operation of this planetary gear set, their attention is directed to the aforementioned US patent wherein a complete description of the operation and the interchange of friction devices is discussed. For the purposes of this disclosure, it will be satisfactory to recognize that when the planetary transmission is operating in the second, third, fourth, fifth or sixth forward drive ratio, full engagement of the friction brake 14 will cause a significant lockup within the planetary gear arrangement. Thus, it should be understood that controlling the friction brake 14 in slipping engagement, when one of the other forward drive ratios is fully engaged, will cause a retardation in the rotation of the output shaft 30 resulting in retardation of vehicle speed.

The hydraulic retarder 26 includes a rotary member 40 connected with the hub 28 which is operable to circulate fluid between vane members found on rotary member 40 and vane members found on a stationary housing 42. Fluid circulation between the vane members on these components 40 and 42 results in power absorption and retardation of the speed of the output shaft 30 which therefore results in retardation of vehicle speed. It should be appreciated that hydraulic retarders, such as 26, will increase in efficiency as the rotary speed of rotary member 40 increases. The operation of hydraulic retarders is a well known phenomenon, so that a more thorough dissertation of the operation thereof is not needed here.

The curves shown in Figure 3 represent friction braking, hydraulic braking and total braking torque. The friction braking is a result of the slipping engagement of friction brake 14 when a forward ratio other than first has been selected. The hydraulic braking is the result of the hydraulic retarder 26 churning fluid between the rotary member 40 and stationary housing 42. The total torque is the summation of friction braking and hydraulic braking. By viewing the curves, it will be recognized that hydraulic braking increases rapidly with output speed. The friction braking is shown to decrease with output speed. This decrease in friction braking is provided by controlling the apply pressure which is present at the friction brake 14.

Referring to Figure 2, there is seen a representation of the friction brake 14, the hydraulic retarder 26 and a hydraulic control circuit. The

hydraulic control circuit provides pressurized fluid for the operation of the friction brake 14 and hydraulic retarder 26.

The hydraulic control circuit includes a conventional hydraulic pump 44 which is operable to draw fluid from a reservoir 46 and deliver fluid to a main pressure passage 48. Fluid pressure in the main pressure passage 48 is controlled by a conventional pressure regulator valve 50. The hydraulic pump 44 supplies fluid pressure to a torque converter which is operatively connected to the input shaft of the planetary transmission shown in Figure 1. The hydraulic pump 44 also supplies fluid to a conventional control circuit, not shown, for the transmission which is operable to control the interchange of drive ratios when the vehicle is operating. The torque converter and hydraulic control circuit can be selected from many of the well known devices which are available to provide these services. The hydraulic control circuit preferably has a computer controlled apparatus 53 wherein external signals or parameters such as engine speed, vehicle speed, throttle position, engine temperature and transmission temperature are recorded and utilized as signals to control the operating condition of the planetary transmission. Also, a brake signal is present which is operable to determine when the vehicle retardation through the use of the hydraulic retarder 26 is requested.

The hydraulic pump 44 also supplies fluid pressure to a brake pressure control valve 52 and to a solenoid valve 54. The solenoid valve 54 is a conventional on/off type solenoid which when open delivers fluid through a passage 56 to a retarder control valve 58. The brake pressure control valve 52 supplies fluid pressure to a retardation brake apply passage 60 which is in fluid communication with the friction brake 14 and to a port on the retarder control valve 58.

The brake pressure control valve 52 is a pulse width modulated solenoid control valve. This means that the duty cycle of a solenoid is controlled such that the output pressure, that is the pressure in retardation brake apply passage 60, is controlled as a function of the duty cycle of the solenoid. This permits very accurate pressure control at the output of the solenoid or brake pressure control valve 52.

The torque converter output flow in a passage 62 is communicated with the retarder control valve 58 and also with a retarder pressure control valve 64. The retarder pressure control valve 64 is also a pulse width modulated solenoid type valve such that the output pressure in an output passage 66 can be controlled as a function of the duty cycle of the solenoid. Pulse width modulated solenoid valves are well known and have been proposed for use in a number of transmission control applica-

tions.

The output passage 66 of the retarder pressure control valve 64 is in fluid communication with the retarder control valve 58.

The retarder control valve 58 has a valve body 68 in which is slidably disposed a multi-landed valve spool 70 which is urged upwardly as seen in Figure 2 by a compression spring 72. In addition to the already enumerated passage connections, the valve body 68 is also in fluid communication with a hydraulic retarder feed passage 74, a hydraulic retarder return passage 76, a brake apply passage 78, a pair of cooler passages 80 and 82, and a lube passage 83.

In the spring set position shown, the multi-landed valve spool 70 provides communication between retardation brake apply passage 60 and brake apply passage 78. Retardation brake apply passage 60 is in fluid communication with a secondary piston 84 while the brake apply passage 78 is in fluid communication with a primary piston 86. The primary piston 86 has a larger area than the secondary piston 84. During the normal transmission operation, the brake pressure control valve 52 will be energized whenever the low ratio friction brake 14 is applied. With the hydraulic control circuit as shown, the apply pressure in retardation brake apply passage 60 will be communicated to the secondary piston 84 and also via retarder control valve 58 and brake apply passage 78 to the primary piston 86. Thus, the friction brake 14 will receive full engagement pressure, and will be fully engaged to prevent rotation of the power transmission member 38.

Also in the spring set position of retarder control valve 58, the torque converter output in passage 62 is directed via multi-landed valve spool 70 to cooler passage 80 from which it passes through the cooler and returns to the lube via retarder control valve 58 and lube passage 83. From the lube circuit, the fluid returns to the reservoir 46. When vehicle retardation is requested, the solenoid valve 54 will be energized such that passage 56 will be pressurized. This will cause the multi-landed valve spool 70 to move against the compression spring 72 or downward as it is seen in Figure 2. With this movement, the brake apply passage 78 is connected to exhaust. The hydraulic retarder return passage 76 for the hydraulic retarder 26 is connected to the cooler passage 80, and the cooler (return) passage 82 is connected to the hydraulic retarder feed passage 74. The pressure in the hydraulic retarder feed passage 74 is controlled by the retarder pressure control valve 64. As previously mentioned, this is a pulse width modulated valve and therefore the retarder feed pressure can be varied. The multi-landed valve spool 70 has a cross drilled passage 88 which permits a portion of

the return flow from the torque converter to be distributed to the lube circuit such that transmission lubrication will continue during retardation, however the majority of the fluid from the torque connector is directed through retarder pressure control valve 64 and output passage 66 to the hydraulic retarder feed passage 74.

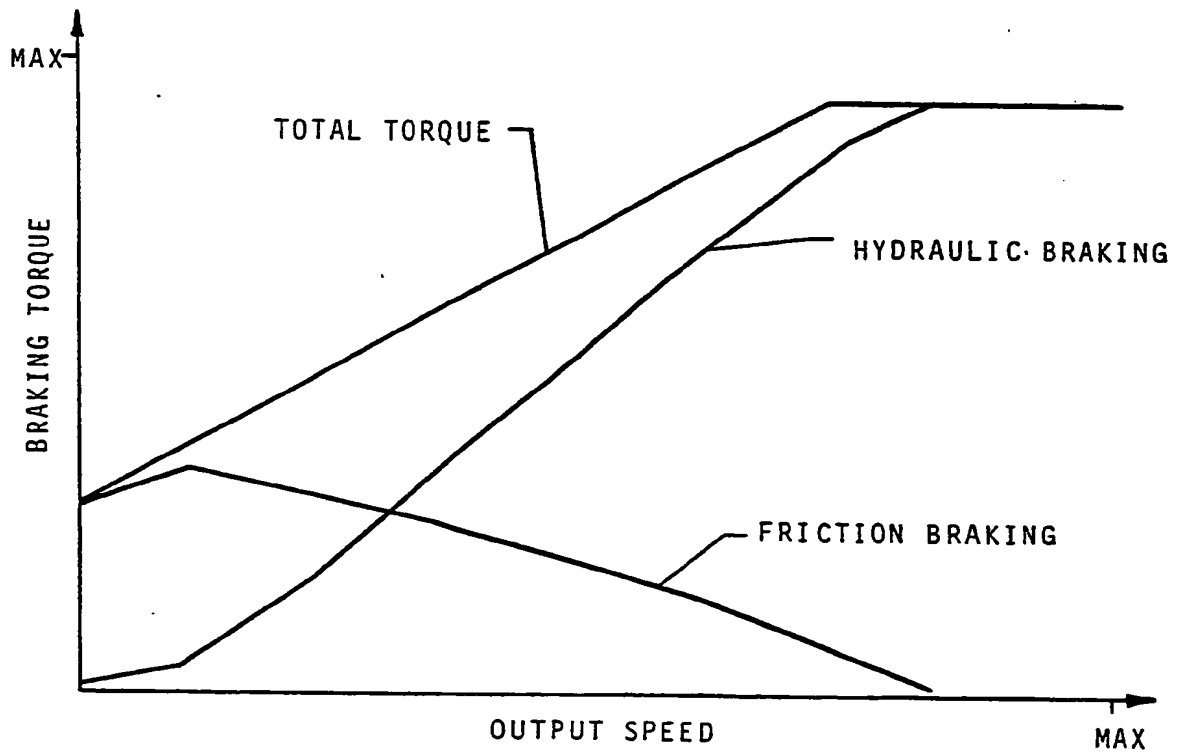
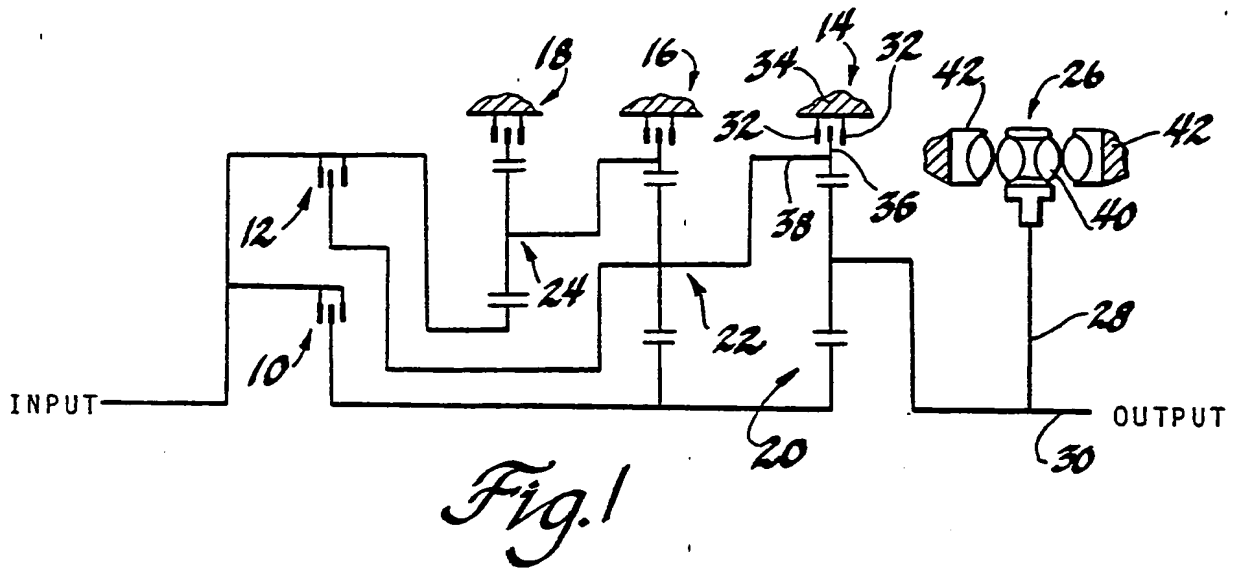
Since the output pressure or the return pressure from the hydraulic retarder 26 is generally greater than the inlet or feed pressure, the recycling of fluid through the cooler to the hydraulic retarder input is permitted. The retardation brake apply passage 60 as previously mentioned is connected with the brake pressure control valve 52. Thus, the pressure within the retardation brake apply passage 60 is controlled by the computer controlled apparatus 53. Thus, the pressure can be controlled as a function of vehicle speed if desired. By providing the pressure as a function of vehicle speed, the engagement or apply pressure in secondary piston 84 can be varied as a function of vehicle speed. Thus the apply force of the friction brake 14 is very easily controlled such that at low vehicle speeds a higher brake force is available. This brake force is controlled such that the friction brake 14 becomes a continually slipping member and does not provide stoppage of the power transmission member 38. Since the friction brake 14 is a continually slipping member during retardation, there is need for cooling and lubrication flow. This is accomplished through cooling means comprising a passage 90 and a restriction 91 which is fed by the hydraulic retarder feed passage 74. This passage 90 is only pressurized when the hydraulic retarder feed passage 74 is pressurized and therefore only makes fluid available to the friction brake 14 for cooling or lubrication when the hydraulic retarder 26 operation is requested. The passage 90 is in fluid communication with one or more nozzle or exhaust passages 92 formed in the housing for the friction brake 14. These exhaust passages 92 are normally covered by a plate 94 which is an extension of the secondary piston 84. As the secondary piston 84 is moved leftward as viewed in Figure 2, the plate 94 is moved leftward with the secondary piston 84 to cause the partial or slipping engagement of the friction discs 32 and plates 36 of friction brake 14. As the plate 94 is moved leftward, the exhaust passages 92 become open through slots 96 such that fluid available thereto can be distributed at the inner surface of the friction discs 32 along the power transmission member 38. The fluid distributed at this point will migrate between the adjacent friction discs 32 and plates 36 such that cooling of the discs and plates is provided during retardation. It is important to maintain the brake plates cooled since the slippage therebetween gives rise to a large amount of heat

generation.

As previously mentioned in Figure 3, the friction braking torque decreases as output speed increases from a minimum value to a maximum value. This friction braking decrease is brought about through the control of the brake apply pressure in retardation brake apply passage 60. Also as set out above, this pressure is controlled by a brake pressure control valve 52 which in turn is controlled by the computer controlled apparatus 53. Thus, the shape of the friction braking curve can be accurately controlled so that it can be efficiently and rapidly reduced when significant hydraulic braking from the hydraulic retarder 26 is available. This pressure can also be controlled in such a manner such that the total output torque curve is given the desired shape.

Claims

1. A control circuit for vehicle retardation for a vehicle having a hydraulically operated multi-ratio power transmission comprising a power output shaft (30), and a selectively engageable friction brake (14) for establishing a low forward drive ratio in the transmission and being operable to retard the vehicle if engaged when another forward ratio has been established; the control circuit comprising a hydraulic retarder (26) operatively connected with the power output shaft (30) of the power transmission for retarding vehicle movement when fluid is admitted thereto; characterised by a pair of fluid pressure operated pistons (84,86) for selectively engaging the friction brake; a control valve (52) for determining the pressure of the fluid directed to both the pistons (84,86); a retarder control valve (58) for directing pressurized fluid to the hydraulic retarder (26), for directing fluid from the hydraulic retarder (26), for disconnecting one of the pistons from the control valve (52) and for connecting the one piston to exhaust; and cooling means (90,91,94) for distributing fluid to the friction brake (14) for cooling thereof only when the other of the pistons is pressurized, the other piston being pressurized to slippingly engage the friction brake (14) when vehicle retardation is desired and when the power transmission is operating in a ratio other than the low forward ratio.



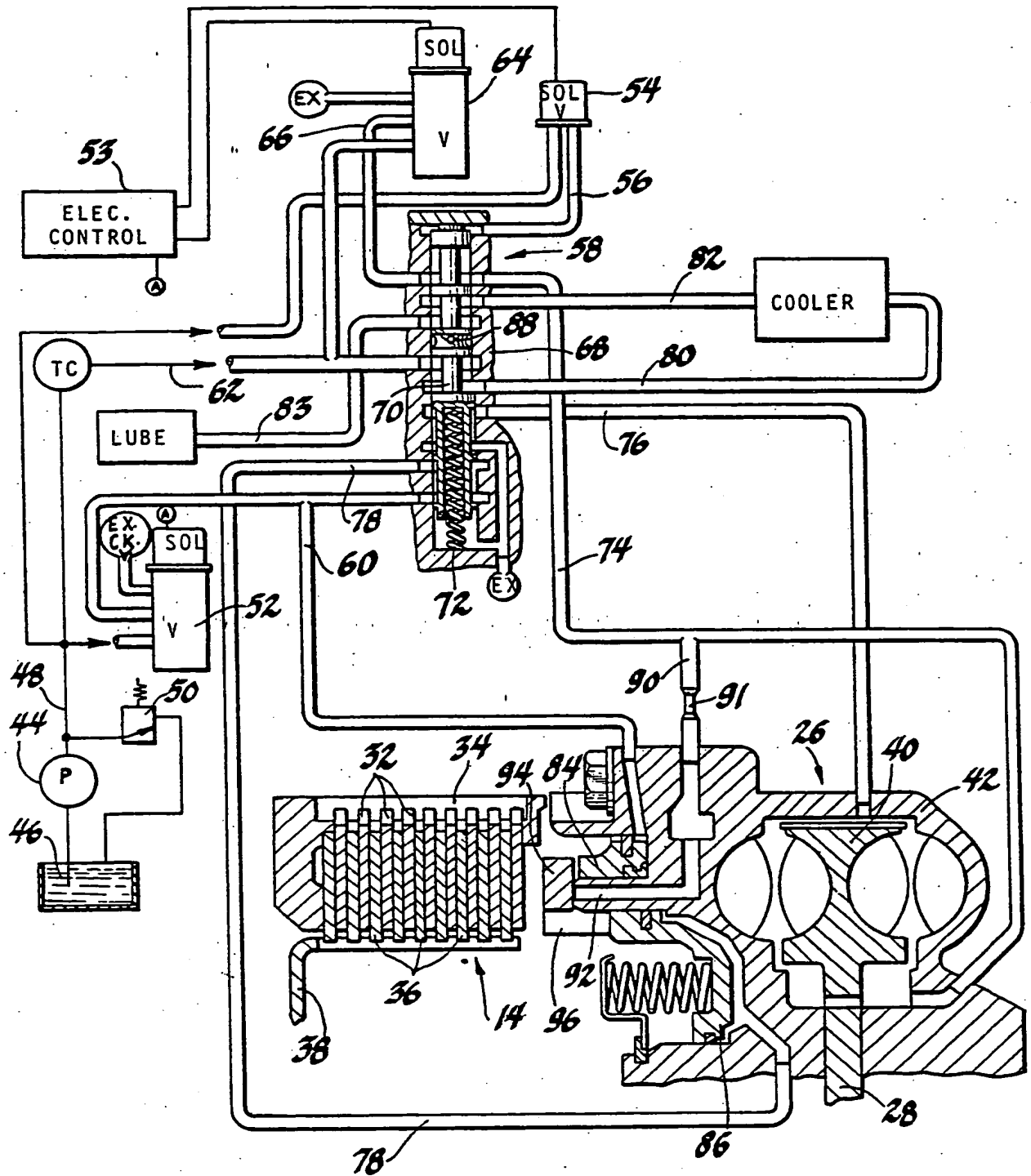


Fig. 2

THIS PAGE BLANK (UPTD)

12 **EUROPEAN PATENT APPLICATION**

21 Application number: 88305067.6

51 Int. Cl.4: B60T 10/02

22 Date of filing: 03.06.88

30 Priority: 23.06.87 US 65350

43 Date of publication of application:
28.12.88 Bulletin 88/52

84 Designated Contracting States:
BE CH DE FR GB IT LI NL SE

68 Date of deferred publication of the search report:
27.12.89 Bulletin 89/52

71 Applicant: **GENERAL MOTORS CORPORATION**
General Motors Building 3044 West Grand
Boulevard
Detroit Michigan 48202(US)

72 Inventor: **Fuehrer, Reece R.**
51 North 200 East
Danville Indiana 46122(US)
Inventor: **Won, Harvey Donald**
786 Winter Court
Carmel Indiana 46032(US)

74 Representative: **Denton, Michael John et al**
Patent Section - Luton Office (F6) Vauxhall
Motors Limited P.O. Box 3 Kimpton Road
Luton Bedfordshire LU2 0SY(GB)

54 **Control circuit for vehicle retardation.**

57 A control circuit for vehicle retardation has a hydraulic retarder (26) and a slipping friction brake (14) operating in concert to retard the forward motion of the vehicle. The hydraulic retarder is an output driven device which is operable to absorb increasing energy as the vehicle speed increases. The friction brake is normally a ratio control reaction brake for the lowest forward drive ratio of the vehicle transmission. The friction brake has an additional apply piston (84,86) which is operable to slippingly engage the friction brake whenever the hydraulic retarder is operable and the vehicle transmission is not in the lowest forward ratio. The control circuit has a control valve (52) which is operable to control the engagement pressure of the additional piston in a manner such that the friction braking effort is higher at low vehicle speeds. This compensates for the lower effectiveness of the output driven hydraulic retarder. The control circuit is also effective to control the operation of the hydraulic retarder, the main piston of the reaction brake, cooling flow to the friction brake and cooling flow from the hydraulic retarder.

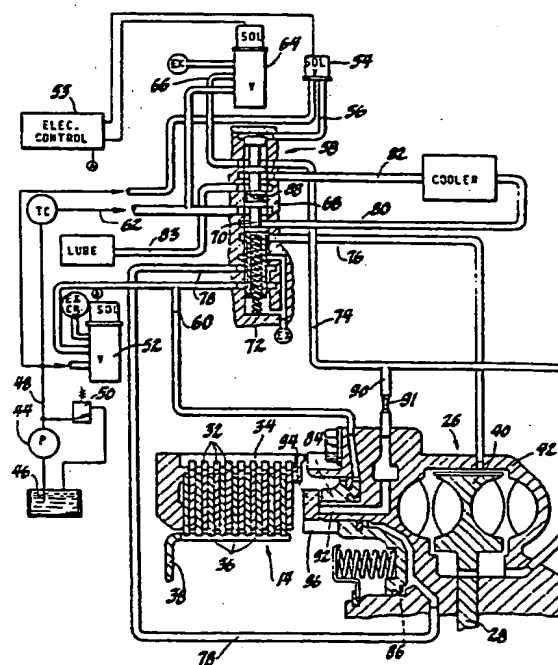


Fig. 2



European Patent
Office

EUROPEAN SEARCH REPORT

Application Number

EP 88 30 5067

DOCUMENTS CONSIDERED TO BE RELEVANT			
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int. Cl.4)
A	GB-A-2 022 743 (GENERAL MOTORS) * figures 1-10 * ---	1	B 60 T 10/02
A	GB-A-2 052 658 (GENERAL MOTORS) * figures 1-10 * ---	1	
A	US-A-4 480 728 (K. A. BAILEY et al.) * figures 1-5 * ---	1	
D,A	US-A-4 070 927 (J. C. POLAK) ---		
D,A	US-A-4 630 507 (A. KUGLER et al.) -----		
			TECHNICAL FIELDS SEARCHED (Int. Cl.4)
			B 60 T F 16 D
The present search report has been drawn up for all claims			
Place of search BERLIN		Date of completion of the search 06-10-1989	Examiner LUDWIG H J
CATEGORY OF CITED DOCUMENTS X : particularly relevant if taken alone Y : particularly relevant if combined with another document of the same category A : technological background O : non-written disclosure P : intermediate document T : theory or principle underlying the invention E : earlier patent document, but published on, or after the filing date D : document cited in the application L : document cited for other reasons ----- & : member of the same patent family, corresponding document			

EPF FORM 1503 03.82 (P0401)